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# **Structural Design Analysis of Thrust Controlled Solid Motor**

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### Abstract:

Towards demonstration of thrust control in solid rocket, Thrust Controlled Motor (TCM) is configured with Hot Gas Flow Control Valve (HGFCV) to vary throat area for thrust modulation. Prototype TCM is developed for repetitive ground testing to fine tune the propellant formulation and finalize material technologies for high temperature and dynamic sealing applications of HGFCV.

The present report deals with the design and structural analysis of Thrust Controlled Motor (TCM) being developed at RCI/DRDO, HYD. The Thrust Control Motor (TCM) design is carried out as per ASME pressure vessel code for MEOP of 200.0 bar. The flange design is carried out following the Schneider's approach. To meet the functional requirements, igniter, nozzle end dome and nozzle casing has been configured with grooves and 'O'-Ring dimensions are finalized keeping its dialation more than 15% of 'O'-Ring Diameter. Finite element analysis is carried out for the above design to verify the stress levels are within allowable limits and also dialation expected due to internal pressure. The flange joint modeled using contact element method to verify opening of 'O'-Ring due to internal pressure and applying prestress through thermal stress values to verify stress level in bolt.

The result indicates that the maximum stresses and dialation are within allowable limits. Hence the motor design is meeting the ground test requirements. The design of motor is updated using the 3D structural analysis results and drawings are finalized based on the design analysis.

### Nomenclature:

i vomenciature.				
Α	=	Minimum area of the bolt		
Ac	=	Effective cross sectional area of the		
flange	undergoi	ng deformation		
<b>b</b> <sub>max</sub>	=	Half distance between PCD and Outer		
Diamet	er			
С	=	Spring Constant of Complete joint		
D	=	Motor Outer Diameter		
Ε	=	Weld Efficiency		
$\mathbf{h}_{\mathbf{g}}$	=	Half distance between PCD and center		
of 'O' i	ring			
Р	=	Maximum Effective Operating		
Pressur	e			
S	=	Maximum Allowable Stress		
L	=	Crown radius		
r	=	Knuckle radius		
Ν	=	No. of bolts/mm of bolt circle		
circum	ference			
t	=	thickness of shell / flange / dome		
$\sigma_{\rm b}$	=	Preload stress		
$\sigma_{bolt}$	=	Yield Strength of bolt		
<b>R</b> <sub>m</sub>	=	Mean Radius		
1	=	Half distance between PCD and mean		
radius				

#### **Abbreviations:**

=	Three Dimensional			
=	American	Soc	iety	of
gineering	3			
=	Centre	For	Aerospa	ace
ces				
=	Computer	Aided	Design	&
=	Computer		Aid	led
=	Dual Thrus	t Motors	5	
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DRDO	=	Defense		Research
Development Organization				
EPP	=	Effective Pivot Point		
FEM	=	Finite Element Method		od
GGP	=	Geometric I	Pivot Poi	nt
HVT	=	High Veloc	ity Trans	ient
ISRO	=	Indian S	Space	Research
Organization				
PCD	=	Parametric Centre Diameter		
PGAD	= Pr	rogramme Air Defence		
RSRM	=	Redesigned	Solid	Rocket
Moor				
RCI	=	Research Centre Imarat		
SRM	=	Solid Rocket Motor		
SITVS	=	Secondary	Injectio	n Thrust
Vector Control	l			
TVS	=	Thrust Vector Control		

### **Introduction:**

Controllable thrust propulsion technology is being explored by missile industry as a solution to the propulsion requirements for the next generation tactical missiles. Modulating thrust in solid propulsion is one approach for achieving this goal.

Using Thrust Controller (TC) a conventional solid propellant rocket motor can provide variable thrust levels, providing the capability to decrease missile flight time to target, or to increase maximum range capability.Development of Non-Al propellant formulations is in progress at IIT, Madras as part of CARS project of PGAD/RCI. There is a requirement of small scale hot tests towards development of non-Al composite propellant system towards development of minimum smoke propellant for Programme 'AD' missile applications.

The present report deals with the design and structural analysis of thrust controlled test motor for repetitive ground testing of non-Al composite propellant formulations being developed at IIT,M / HEMRL. The design of motor is updated using the structural analysis results and drawings are developed accordingly.

### **Configuration & Specification of Motor:**

Developmental tests are carried out in straight nozzle configuration accommodating changes in burn rate and grain configurations. The configuration of motor with straight nozzle is shown in Fig. 5.22 It consists of straight nozzle with attached to the motor chamber at nozzle end. The specifications of the motor and geometrical details are given below.

### **Motor Specifications:**

Total Impulse :17500.0  $\pm$  1500 Kgf-s Nominal Thrust:2000.0 Kgf Nozzle system :one straight nozzle Nominal burning time :8.00 s Temperature range:  $-20^{\circ} \text{C} - +55^{\circ} \text{C}$ Altitude of test motor : Sea level Propellant :Non-Al composite Ignition System: Pyrotechnic Propellant Weight:  $\approx 60 \text{ kg}$ Space Envelope Motor Outside diameter: \$650mm  $\approx 800 \text{ mm}$ Motor length :

### Motor Design:

The L/D of the motor casing is less than 1.0. The motor casing is designed as per ASME pressure vessel code [1]. Flange design at head end and nozzle end are designed using Schneider's approach. Flange thickness is calculated using flat plate closure formula as per ASME pressure vessel code. Tori-spherical dome configuration is selected of easy fabrication for head end and nozzle end domes and thickness is calculated using ASME pressure vessel code [1].

### **Material Properties:**

High strength 15CDV6 steel material is chosen for motor casing and nozzle due to its high strength to weight ratio, availability and well established fabrication procedure. The specification of the material is given below.

Chemical Composition of 15CDV6 (Wt. %): Cr = 1.5 Max.



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Mn	=	1.1	Max.
Mo	=	1.1	Max.
V	=	0.3	Max.
С	=	0.18	Max.
Ni	=	0.25	Max.
Cu	=	0.2	Max.
Si	=	0.2	Max.
Р	=	0.02	Max.
S	=	0.015	Max.

### **Mechanical Properties:**

Property	(Tangential)	(Transverse)
Tensile Strength (MPa)	1080 (min.)	1020 (min.)
Yield Strength (MPa)	930 (min.)	880 (min.)
% elongation (on 50mm) min.	10	10
Hardness	290-360	290-360
Impact toughness (J) 2mm charpy 'U' notch)	60 (min.)	60 (min.)
Plain Strain fracture toughness (MPa)	80	80

### **Factor of Safety Selection Criteria:**

The following safety factors have been chosen for the design of rocket motor for repetitive ground tests.

On Yield	= 1.5;	Allowable = 587 MPa
On UT	= 2.0;	Allowable = 510 MPa

#### **Operation of Hot Gas Flow Control Valve:**

The configuration of the HGFCV is shown in Fig.1. It consists of hot gas generator based on solid propellant attached with convergent-divergent nozzle. The valve pintle is arranged in throat region of the nozzle and area is varied by opening/closing the pintle using BLDC actuator as shown in the figure. Closed loop control of pintle position varies the throat area in real time achieving flow control. Hot gasses are generated due to burning of the solid propellant at the propellant surface injected into the core region and enter nozzle convergent and divergent regions. Expansion of high pressure and high temperature hot gasses from nozzle inlet to outlet generates high velocity and momentum. The reaction force due to expulsion of gasses generates thrust force.

Thrust is computed at exit plane as an integration of the momentum due to high velocity and pressure difference as follows:

 $\int \{ mV + A(P_e - P_a) \} dr \dots (1)$ 

Varying the flow rate changes the exit momentum thereby changing the thrust force.

Flow rate is varied altering the equilibrium chamber pressure between inlet mass flow due to propellant combustion and out let flow through the choked nozzle.

The inlet flow rate due to hot gas generation is function of burn rate and surface are of the grain as per the following equation:

$$m p = A b \rho r \dots (2)$$

Propellant burn rate is function of chamber pressure and it is given by as St. Robert's law:

$$r.=aP_c^n \dots \dots (3)$$

where *r* is the burn rate, *a* is the pre-exponential factor,  $P_{c}$  is the chamber pressure, and *n* is the pressure exponent.

The outlet flow rate of the choked nozzle based on 1-D isentropic flow is proportional to the throat area and chamber pressure as follows:

The equilibrium pressure of the system is established as per the following continuity equation between inlet flow (2) and out let flow (4).



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 $\frac{dP_c}{dt} = \frac{RT_g}{V} \left( A_b \rho a P_c^n - \frac{P_c A_t}{C^*} \right)^{-1}$ 

Varying the throat area establish new equilibrium pressure and then changes the flow rate through the nozzle. The principle is based on the controlling the throat area using the pintle valve to change the equilibrium chamber pressure to vary the flow rate achieving the thrust modulation. Moving the pintle outwards increases throat area decreasing the chamber pressure and flow rate. Moving the pintle into the throat region decreases the throat area and increases the chamber pressure increasing the flow rate of the thrusters. Closed control of pintle position using BLDC actuator achieves the thrust modulation in real time.

#### Geometry:

Solid works 12.0 is an interactive Computer- Aided Design and Computer Aided Manufacturing system. The CAD functions automate the normal engineering, design and drafting capabilities found in today's manufacturing companies. The CAM functions provide NC programming for modern machine tools using the Solid works 12.0 design model to describe the finished part. Solid works 12.0 functions are divided into "applications" of common capabilities. These applications are supported by a prerequisite application called "Solid works 12.0 Gateway"

Solid works 12.0 is fully three dimensional, double precision system that allows to accurately describing almost any geometric shape. By combining these shapes, one can design, analyze, and create drawings of products



Fig.4.1.Geometric model of Thrust Controlled Solid Motor



Fig.4.2 3DGeometric model of Thrust Controlled Solid Motor (Cross Section)

### **Results and Discussion:**

**Meshing:** Finite Element Discretization or Meshing is the process used to "fill" the solid model with nodes and elements, i.e., to create the FEA model. Remember, you need nodes and elements for the finite element solution, not just the solid model. The Solid Model in CAD does NOT participate in the finite element solution



Fig (1): 2d Mesh Model

#### 2D Loads And Boundary Conditions:

- The analysis has been carried out with pretension ~61.58°C applied to Bolts.
- Upper area constrained in all D.O.F direction.
- Applied pres on inter area of body



Fig (2): 2d Loads & Boundary Condition Applied To Motor

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Fig (3) : VonMises Stress Plot



Fig (5.5): contacts mesh



Fig (5.6): VonMises Stress Plot (minimum position)



Fig (5.7): VonMises Stress Plot (maximum position)



Fig (5.8): Head End Igniter Configuration



Fig (5.9): Head End Dome Configuration



Fig (5.10): Motor shell Configuration



Fig (5.11): Nozzle End Dome Configuration



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Fig (5.12): Nozzle End Flange Configuration



Fig (5): Nozzle Exit End Flange Configuration

REGION	Von-Mises	FOS on
Region-I	16.605 (74.29)*	2.76
Region-II	30.831 (55.211)*	1.48
Region-III	31.055	1.47
Region-IV	16.668	2.75
Region-V	13.21(39.72)*	3.47
Region-VI	17.23(55.59)*	2.66

Allowable stress on weld = 45.9 Kg/mm2 Table 1.0. FOS and stresses at various locations of Motor (Excluding localized stresses).

\* The values shown in brackets are highly localized. These are not considered for calculation of FOS.



### **3D Loads And Boundary Conditions:**

- The analysis has been carried out with pretension ~61.58°C applied to Bolts.
- Upper area constrained in all D.O.F direction.
- Applied pres on inter area of body



Fig (5.15): 3D: FE Mesh Model of Motor



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Fig. 5.21: VonMises Stress Plot (minimum Position)



**3D Nozzle End Flange Configuration** 

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**3D Nozzle Exit End Flange Configuration** 

REGION	Von-Mises stress in (Kg/mm² )	FOS on Yield
Region-I	27.4338	1.67
Region-II	24.669 (55.316)*	1.86
Region-III	33.093	1.38
Region-IV	20.8112	2.20
Region-V	16.77(37.1827)*	2.73
Region-VI	16.6057	2.76

Allowable stress on weld = 45.9 Kg/mm2 Table 2.0. FOS and stresses at various locations of Motor (Excluding localized stresses). \*The values shown in brackets are highly localized. These are not considered for calculation of FOS.



**Regions of 3D Motor for stress reporting** 

### **Conclusions and Future Scope of Work**

The straight nozzle motor design is carried out as per ASME pressure vessel code for MEOP of 200.0 bar.

The flange design is carried out following the Schneider's approach. To meet the functional requirements, igniter, nozzle end dome and nozzle casing has been configured with grooves and 'O'-Ring dimensions are finalized keeping its dilation more than 16% of 'O'-Ring Diameter. Finite element analysis is carried out for the above design to verify the stress levels are within allowable limits and also dilation expected due to internal pressure. The flange joint modeled using contact element method to verify opening of 'O'-Ring due to internal pressure and applying prestress through thermal stress values to verify stress level in bolt. The result indicates that the maximum stresses and dilation are within allowable limits. Hence the motor design is meeting the ground test requirements.

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