

## Performance Improvement of a Micro Channel Heat Sink

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### ABSTRACT:

*Thermal management of components is the most important consideration for electronic cooling. The microchannel heat sinks have been developed by many researchers in recent years. Numerous literatures have been published on fluid flow and heat transfer characteristics of microchannel. Tuckerman and Pease have demonstrated a heatsink with micro channel. In this research they found that microchannel heat sinks can extract more heat and are preferred for chip cooling in VLSI circuit. Have manufactured a counter flow microchannel heat exchanger by precision cutting. That microchannel heat exchanger is tested with water as working fluid for convective heat transfer. A volumetric heat transfer coefficient of  $0.324 \text{ MW}/(\text{m}^3\text{K})$  was obtained. Have manufactured a trapezoidal shaped microchannel heat exchanger by diamond machining. In that heat exchangers counter flow arrangement is provided.*

*In this present study we will develop a new models of micro channel heat sink which can be manufactured by three layers namely the base, core and top layer, the base will be in contact with the heating source and the fluid will flow through the core, here different geometries of the core are developed and tested using some fem software, 3D models of the components are developed using Catia V5*

### INTRODUCTION

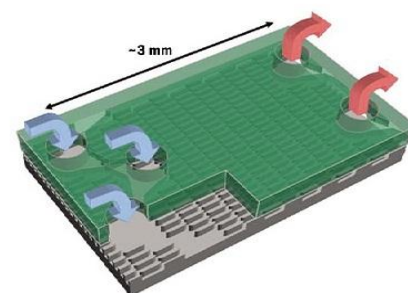
#### Microchannel Heat Sinks

With component heat dissipation levels reaching  $500 \text{ W}/\text{cm}^2$  and beyond, conventional air cooling systems are inadequate for removing excess heat. Research has intensified toward developing more innovative chip cooling techniques. The ultimate goal is to reduce

thermal resistance from the chip junction to ambient, and keep the chip's junction temperature as low as possible.

For high performance CPUs, graphics cards, power amplifiers and other devices, air-cooling has proven ineffective at dissipating high heat fluxes. Heat transfer methods such as heat pipes, vapor chambers, Nanomaterials, liquid cooling and miniature refrigeration systems have been attracting more interest.

Liquid-cooled microchannel heat sinks and coolers have been shown to be a very effective way to remove high heat load. A large heat transfer coefficient can be achieved by reducing the channel hydraulic diameter. In a confined geometry the small flow rate within microchannels produces laminar (smooth) flow, which results in a heat transfer coefficient inversely proportional to the hydraulic diameter. In other words, the narrower the channels in the heat sink, the higher the heat transfer coefficient.



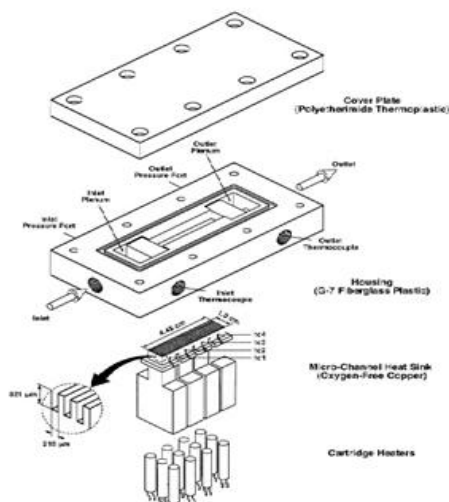
**Fig.1 Micro Channel Cooler Showing Heat Exchanger Zones**

Forcing a fluid through a greater number of small channels vs. a smaller number of larger channels increases the level of heat transfer from the hot source,

e.g. chips, cards, into the flowing liquid. There are more channels for a small hydraulic diameter than large ones. So the heat sink with a smaller hydraulic diameter has better overall thermal performance.

The use of microchannels as a viable cooling solution was first proposed in 1981 by Tuckerman and Pease, who designed and tested an integral, water-cooled heat sink by etching microscopic channels 50  $\mu\text{m}$  wide and 300  $\mu\text{m}$  deep on the silicon substrate. They reported achieving a high heat flux of 790  $\text{W}/\text{cm}^2$  with a temperature rise of 71°C above the inlet water temperature. Tuckerman's work was well received by the electronics community, and many extensive studies have since been conducted on different aspects of microchannels in electronic cooling.

Even though the use of microchannel devices seems very promising, they require a significant amount of power to push the fluid through the channels in high heat flux applications. One solution is to use convective boiling heat transfer and two-phase flow in the microchannels. Use of boiling heat transfer could improve the efficiency of microchannels in two ways: it reduces the pumping power required to push the fluid through the channels, and at its time of phase change, the boiling coolant absorbs energy from the hot surface of the microchannel heat sink, substantially increasing the heat transfer coefficient.



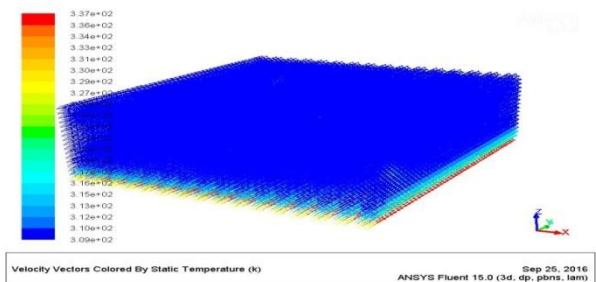
**Fig.2 Microchannel Heat Sink Test Module**

The heat transfer ability attained with two-phase flow and boiling inside microchannels is limited only by its critical heat flux (CHF). The CHF of boiling at a surface refers to the heat flux at which the surface heat transfer coefficient suddenly drops. Exceeding the CHF will lead to a sudden jump in surface temperature which will result in the failure of an electronic component.

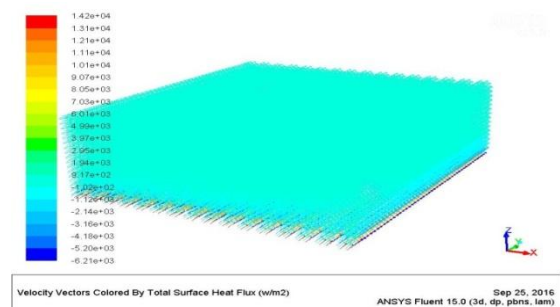
While using microchannels to cool electronics is attractive because of their very high heat transfer coefficient, their use and implementation is very challenging. Factors such as cost, manufacturing, pump selection, filtering requirements to prevent channel clogging, and space constraints must be evaluated before one commits to using microchannels in a system. The research in this area is ongoing and the implementation of this concept will become more widespread once the practical difficulties mentioned above are resolved.

## ANALYSIS

CFD analysis of basic model microchannel heat sink with H14



**Fig.3 CFD analysis showing static temperature for H14**

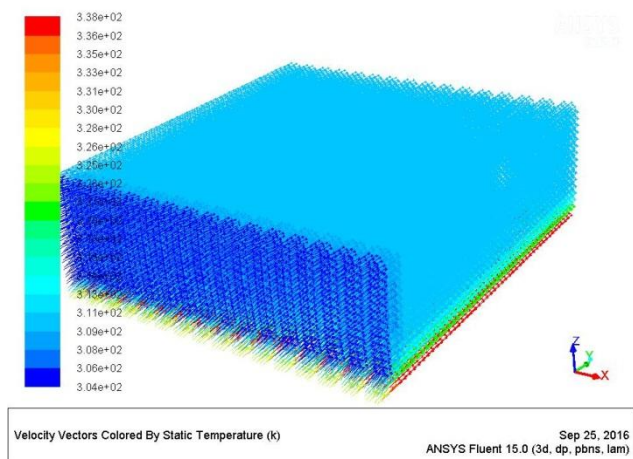


**Fig.4 CFD analysis showing Total surface heat flux for H14**

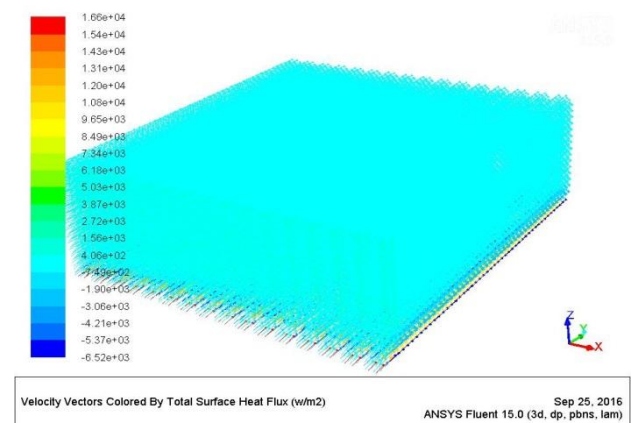
Mass Flow Rate (kg/s)	
Heater	0
Inlet	1.3328e-08
Interior-_msbr	-6.2589631e-07
Outlet	-1.329901e-08
wall-_msbr	0
Net	2.8989808e-11
Total Heat Transfer Rate (w)	
heater	0.16328338
inlet	-0.0037143458
outlet	-0.00017425759
wall-sbr	-0.15939491
Net	-1.315419e-07

Mass Flow Rate (kg/s)	
heater	0
inlet	1.3327998e-07
interior-msbr	-6.2593544e-06
outlet	-1.3308215e-07
wall-msbr	0
Net	1.9782907e-10
Total Heat Transfer Rate (w)	
heater	0.17012226
inlet	-0.015155879
outlet	-0.0018700995
wall-msbr	-0.15308909
Net	7.1940367e-06

### CFD analysis of basic model microchannel heat sink with copper



**Fig.5 CFD analysis showing static temperature for copper**



**Fig.6 CFD analysis showing Total surface heat flux for copper**

### TABLES

#### Thermal Analysis Report

Thermal analysis of heat sink with h14	Temperature(°C)		Total Heat Flux(W/Mm²)		Directional Heat Flux(W/mm²)	
	Minimum	Maximum	Minimum	Maximum	Minimum	Maximum
Basic Model	70.629	73.76	5.48e-04	0.77682	-1.94e-02	1.94e-02
Model 2	72.429	75.703	1.41e-04	1.2315	-0.21565	0.36458
Model 3	61.83	66.813	1.89e-03	1.6663	-0.37144	0.41955
Model 4	94.802	101.13	1.18e-03	2.2256	-1.0153	0.89066

**Table.1 Steady state thermal analysis results for different models of microchannel of H14**

Thermal Analysis Of Heat Sink With Copper	Temperature(°C)		Total Heat Flux(W/Mm²)		Directional Heat Flux(W/Mm²)	
	Minimum	Maximum	Minimum	Maximum	Minimum	Maximum
Basic Model	73.495	75.188	1.69e-03	0.7455	-0.43861	0.43861
Model 2	73.384	75.301	1.27e-03	1.2056	-0.4745	0.47744
Model 3	62.419	65.31	4.31e-04	1.6825	-0.37476	0.42165
Model 4	95.64	99.261	7.14e-04	2.2241	-0.92725	0.94367

**Table.2 Steady state thermal analysis results for different models of microchannel of Copper**

#### CFD Analysis Report

Mass Flow Rate (Kg/S)	Basic Model	Model 2	Model 3	Model 4
H14	2.90e-11	4.69e-12	3.35e-12	5.49e-12
Copper	1.98e-10	4.69e-12	3.35e-12	5.49e-12

**Table.3 CFD analysis-Mass Flow Rate results for different models of microchannel.**



Total Heat Transfer Rate (W) From Heater	Basic Model	Model 2	Model 3	Model 4
H14	0.163283	0.1685611	0.2109391	0.15963343
Copper	0.170122	0.1664593	0.22367207	0.16732258

**Table.4 CFD analysis-Heat Transfer Rate results for different models of microchannel.**

## CONCLUSION

Here we conducted thermal and CFD analysis on four different heat sink models in which first one is a basic model is presently used in cooling applications in computer and other electronics, later designs are developed from the basic models to improve the cooling effect

From the study we can observe that even though we increased the surface area in model two the temperature is slightly raised but these temperatures are much reduced in in model three and then increased in fourth model due reduction in surface area tough we know the reason this attempt is made to check whether turbulence can play

Hear in both studies model 3 is doing better in thermal analysis model three recorded 9.4% less temperature when made with Aluminium and 13.13% less temperatures when made with copper similarly in CFD analysis heat removed from the source is 22.5% more in Aluminium model and 31.47% in copper model

## FUTURE SCOPE

This study can be carried forward by developing more heat sink models and cooling fluids as these are micro channels we can't use all kinds of fluids

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