

Causes of Wall Thinning of a Spinning Component & Mathematical Modeling of Wrinkling

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Abstract

Over the most recent two decades or somewhere in the vicinity, flow forming has been developing as a metal shaping procedure for the generation of engineering parts in little and medium bunch amounts. One of the most earliest used technique for chip-less development is Metal spinning. Customarily, in this procedure a tubular work-piece is hung on a mandrel and prolonged, through thickness diminishment, by pivotally moving rollers. High exactness, thin walled and tubular segments are customary flow shaped items. The power and force necessities are much lessened as compared to the press forming, due to the incremental process and also at a time just a little volume of metal is deformed. In this paper, impact of roller way and feed ratio on a spun sample's wall thickness and also numerical model of wrinkling failure were done. There was unique blank thickness due to high feed ratio additionally prompt material failure and unpleasant surface wrap up.

Keywords- chip-less, impact of roller, material failure, mandrel

INTRODUCTION

A process of plastic shaping, in which metal sheet was formed by squeezing the metal onto a pivoting mandrel utilising a roller device called Metal spinning procedure. It is broadly utilised to make spherical hollow metal items. this procedure is appropriate for moderate generation with a large variety of items and it is also most effective in item advancement and

prototyping. Since there is just a single mandrel is required that costs considerably less than deep drawing. The movement of forming roller is gradually while the mandrel and material quickly rotate. Consequently the results of metal spinning have been innately restricted to axisymmetric shapes that have round cross areas around the axis of rotation. However, there is a large demand for non-axisymmetric items produced by metal spinning procedure which have cross sections like polygonal, elliptic etc. This process is used at large scale if it is possible to produce non-axisymmetric items. Amano and Tamura and Gao et al. introduced spinning machines for non-axisymmetric elliptical cross section items. Shindo et al. prevailing in metal spinning of funnels with unusual or slanted axes.

Nevertheless, for each shape every technique demands an extraordinarily outlined spinning gadget.

The Metal spinning alludes to a gathering of three procedures which are traditional spinning, tube spinning and shear spinning. All these three procedures having similar feature is that they permit rotationally symmetric and hollow parts production. The principle difference between these processes can be seen on Wall thickness of the final product. In traditional spinning, the wall thickness remains almost consistent all through the procedure, because of this; the last wall thickness of the shaped part is equivalent to the blank. Whereas, in the tube spinning and shear spinning the wall thickness is decreased. In tube spinning, the last thickness is characterized by the expansion long of the work piece.

In shear spinning, the part thickness is defined by the angle between the component wall and rotational axis.

Parts can be shaped in a solitary stride or various strides whereas it is done only in a solitary stride in case of shear spinning.

Mathematical Model For Wrinkling Failure

We already know that rolling differs from spinning due to the action of forces during processes occur. A rolling process is a 2-dimensional deformation system but a spinning is 3-dimensional deformation system.

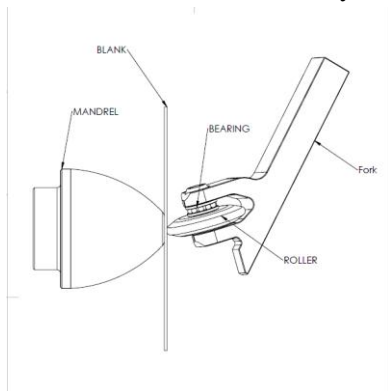


Fig. 1- Spinning experiment setup

As shown in experimental setup blank acts as a column (condition- one end free and one end is fixed) according to the condition of strength of material (subject).

The buckled blank condition of deflection surface is illustrated by the dotted curves as shown in fig.2

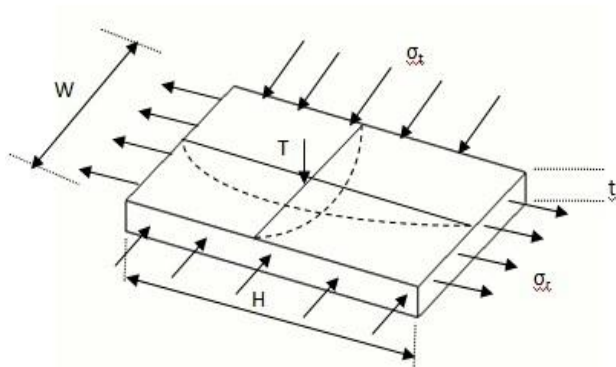


Fig. 2- Buckle condition of column

Timoshenko grants the buckled state of declination surfaces as

$$\delta = \psi \sin \frac{\pi x}{w} \sin \frac{\pi y}{H} \dots \dots \dots (1)$$

Where ψ represents the declination of the blank's buckling surface.

Also, bending energy can be computed by (Timoshenko)

$$\xi_b = \frac{1}{2} F \int_0^H \int_0^W \left\{ \left(\frac{\partial^2 \delta}{\partial x^2} + \frac{\partial^2 \delta}{\partial y^2} \right)^2 - 2(1 - \mu) \left[\frac{\partial^2 \delta}{\partial x^2} \frac{\partial^2 \delta}{\partial y^2} - \left(\frac{\partial^2 \delta}{\partial x \partial y} \right)^2 \right] \right\} dx dy \dots \dots \dots (2)$$

By putting the value of eq.(1) in eq.(2) we get

$$\xi_b = \frac{wH}{8} F \psi^2 \left[\frac{\pi^2}{w^2} + \frac{\pi^2}{H^2} \right]^2 \dots \dots \dots (3)$$

where, F represents the blank's flexural rigidity

$$F = \frac{Et^3}{12(1 - \mu^2)} \dots \dots \dots (4)$$

Where,

E = Young's modulus

t = wall thickness

μ = Poisson's ratio

Gere grants elastic bending theory in 2001 which can be enhanced for covering plastic bending by changing E with diminished modulus E_0

$$E_0 = \frac{4EE_T}{[\sqrt{E} + \sqrt{E_T}]^2} \dots \dots \dots (5)$$

Where

E_T = stress strain slope curve at a specific estimation of strain in the plastic zone

The energy because of tangential stress σ_t can be computed by

$$\xi_t = \frac{1}{2} \int_0^H \int_0^W \sigma_t t \left(\frac{\partial \delta}{\partial x} \right)^2 dx dy \dots \dots \dots (6)$$

Putting the value of eqn. (1) in eqn. (6)

We get,

$$\xi_t = \frac{H \phi^2 \pi^2 t \sigma_t}{8w} \dots \dots \dots (7)$$

the energy because of radial stress σ_r can be computed by

$$\xi_r = \frac{1}{2} \int_0^H \int_0^W \sigma_r t \left(\frac{\partial \delta}{\partial y} \right)^2 dx dy \dots \dots \dots (8)$$

Putting the value of eq.(7) in eq.(8) we get

$$\xi_r = \frac{w\phi^2\pi^2\sigma_r}{8H} \dots\dots\dots(9)$$

As indicated by Timoshenko and woinowsky-krieger, the declination coming about because of a concentrated load T can be communicated as

$$\delta = \frac{\theta TH^2}{F} \dots\dots\dots(10)$$

Where θ represents the numerical factor, value of θ depends on the ratio of $\frac{w}{H}$ and is illustrated by

| | | | | | | |
|---------------|---------|---------|---------|---------|---------|----------|
| $\frac{w}{H}$ | 1.0 | 1.1 | 1.2 | 1.4 | 2 | ∞ |
| θ | 0.01160 | 0.01265 | 0.01353 | 0.01484 | 0.01651 | 0.01695 |

The lateral energy ξ_L because of concentrated loading T which can be computed as,

$$\xi_L = \int_0^\phi T d\delta = \frac{F\phi^2}{2\theta H^2} \dots\dots\dots(11)$$

the critical state reaches when

$$\xi_r + \xi_b = \xi_t + \xi_l \dots\dots\dots(12)$$

where, ξ_r and ξ_b represents the energies which have a tendency to reestablish the harmony, while ξ_t and ξ_l represents the energies that prompt wrinkling failures.

Putting the above value in eq. (12)

$$F \left[\frac{\pi^2}{w^2} + \frac{\pi^2}{H^2} \right]^2 + \frac{\pi^2\sigma_r t}{H^2} = \frac{\pi^2\sigma_t t}{w^2} + \frac{4F}{\theta w H^3} \dots\dots\dots(13)$$

Dividing eq. (13) by $\frac{\pi^2 t}{w^2}$ and familiarize notations τ_ϵ and ρ where

$$\tau_\epsilon = \frac{\pi^2 F}{w^2 t} \text{ and } \rho = \frac{w}{H}$$

The wrinkling critical state can be represented as

$$\tau_\epsilon \left[(1 + \rho^2)^2 - \frac{4\rho^3}{\pi^4 \theta} \right] = \sigma_t - \sigma_r \rho^2$$

Consequently scientific model confirms that the radial and critical tangential stress of wrinkling demonstrate on the wrinkled blank's geometry as well as material property.

Effect of Roller Path on Wall Thickness of a Component

The thickness of a component formed during a spinning process is depend upon the type of spinning [conventional spinning, shear spinning or tube spinning] and also the roller path design during spinning process.

We already know that spinning is an three dimensional deforming process. During conventional spinning process Axial force (F_a) is along the mandrel axis and also Axial force is maximum than two simultaneous acting forces during process these are Radial force (F_r) and Tangential force (F_t).

Case -1 Concave Roller Path

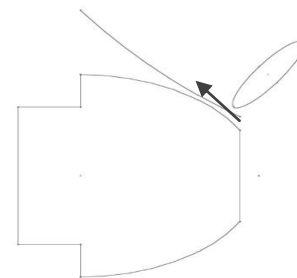


Fig. 3- Concave roller path profile.

During concave roller path profile in conventional spinning process axial force tends to flow the grains of blank (material) in forward direction (in the direction of roller path) and no barrier of flow of grains. As number of passes increases the thickness of blank occurs maximum. As shown in fig. 3 the tip of blank having max distance from the mandrel, this carry maximum bending of the blank.

Case-2 Convex Roller Path

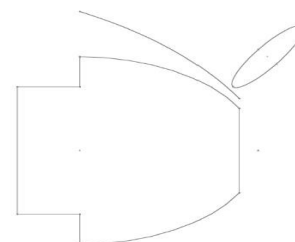
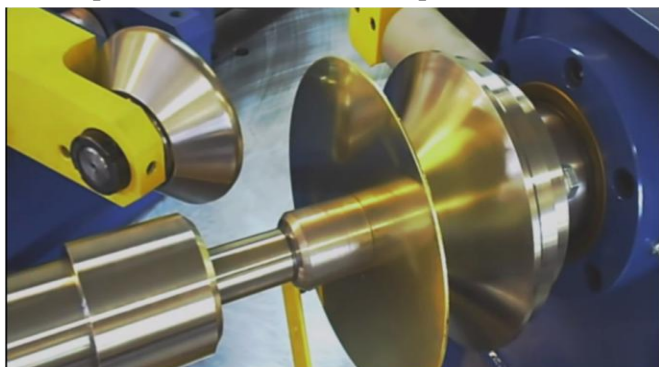
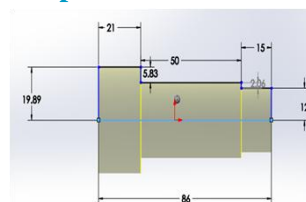
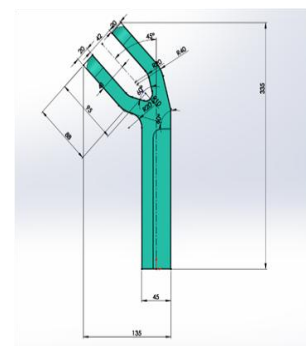
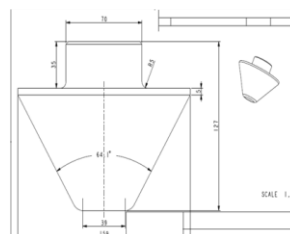


Fig. 4- Convex roller path profile.



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roller path profiles incorporate concave, convex and linear path profile.

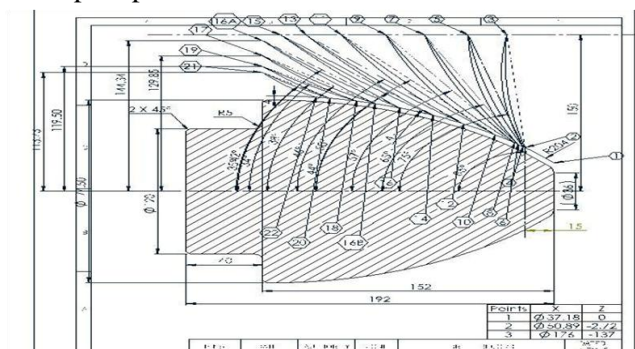


Fig. 7- Trial 1

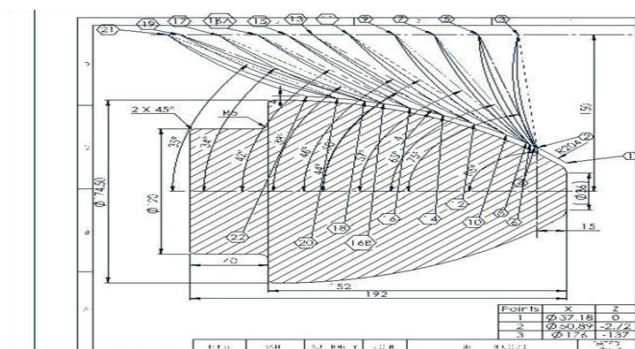


Fig. 8- Trial 2

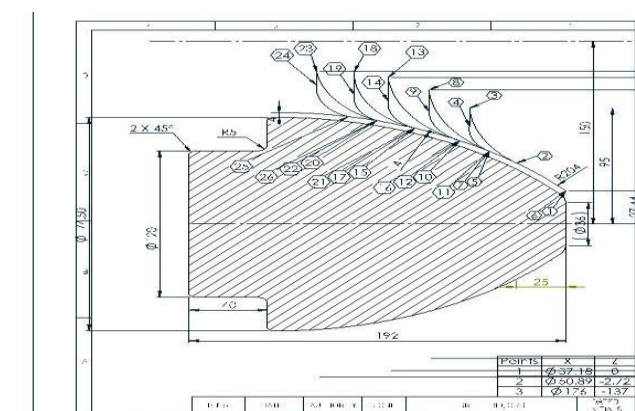


Fig. 9- Trial 3

Verifying experiments have been carried out by using roller path trials 1,2 and 3. The roller path profiles are developed to study the effects of roller path on the wall thickness variation. The experimentally spun samples; their wall thickness variation have been measured and obtain a graph for compare the roller path profiles thickness variation.



Fig. 10- Experimentally spun samples by using different CNC roller paths

Results

Wall thickness

Effect of roller path

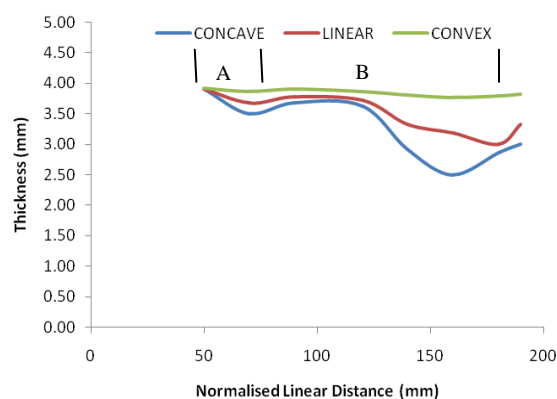


Fig. 11 Thickness reduction in 1st roller pass

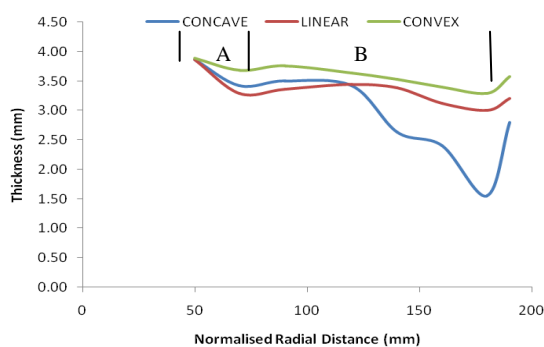


Fig. 12 Thickness reduction in 2nd roller pass

Fig 11 shows the impacts of the profiles of three roller path on wall thickness varieties. it is apparent that the wall thickness diminishes altogether in two zones; Zone A and B . zone A is situated between the backplate's clasped area and the workpiece's last point of contact with the mandrel. zone B is situated between the work piece's last contact point with the mandrel and its contact point with the roller.

As represented in fig 12, after the first forward pass, an sudden decreasing in wall thickness is seen in Region B, particularly in the concave roller path and least decreasing in the convex roller path. As the quantity of passes expands the concave roller path and convex roller path gives practically same thickness lessening in Region A, while straight roller path gives an adjustment in thickness in Region A as correlation with other roller path profile.

Effect of feed ratio

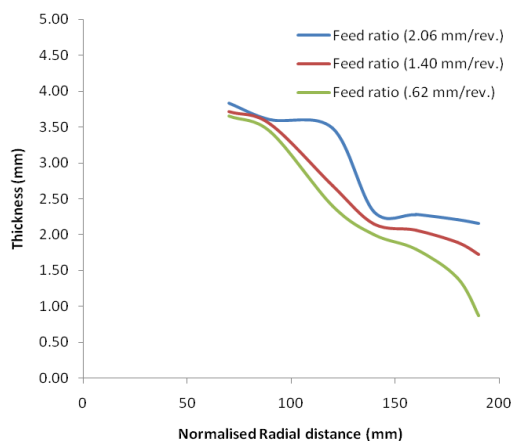


Fig. 13 Thickness reduction due to feed ratio

Fig 13 represents the impacts of feed ratio on the wall thickness distribution of the specimen formed.

Unmistakably, less diminishing of the wall thickness happens if a high feed ratio is enforced. Shearing between the workpiece and roller because of frictional impacts might be one of the primary reasons of the material diminishing. Considering the roller feeds a similar separation amid the spinning procedure, when utilizing a lower feed ratio, the roller will examine the

workpiece with more revolutions, hence prompting higher shearing impacts than spinning at a high feed ratio.

Unmistakably so as to keep up the original blank thickness unaltered, high feed ratio ought to be utilized. In any case, high feed ratio could likewise prompt material failure and rough surface finish.

Conclusion

High feed ratio prompt material failure and rough surface finish. Notwithstanding, high feed ratio additionally help to keep up the original blank thickness unaltered.

Concave roller path profile gives most extreme decrease in wall thickness as correlation with convex roller path profile and straight roller path profile.

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