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Determination of Thermal Performance on Combined Airconditioning and Refrigeration Unit

Madaraboina Prakash

Ellenki College of Engineering and Technology, JNTU, Hyderabad, Telangana, India.

ABSTRACT:

In recent years, the escalating cost of energy has drawn much more attention on improving the energy efficiency of super market operations. In a supermarket refrigeration system consume a large amount of energy in maintaining chilled and frozen food. Meanwhile a HVAC (heating, ventilating and air conditioning) system is used to assure thermal comfort for occupants and suitable climatic conditions for refrigerated cases.

In this thesis, the thermal performance of combined air conditioning and refrigeration unit will be analyzed by CFD. 3D model and assembly of the combined air conditioning and refrigeration unit will be done in Pro/Engineer.

CFD and thermal analysis will be on the unit to determine the thermal performance by varying the refrigerants and materials. Analysis will be done in Ansys.

INTRODUCTION INTRODUCTION TO AIR CONDITIONER

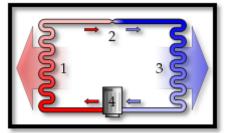
An air conditioner (often referred to as **AC**) is a home appliance, system, or mechanism designed to dehumidify and extract heat from an area. The cooling is done using a simple refrigeration cycle. In construction, a complete system of heating, ventilation and air conditioning is referred to as "HVAC". Its purpose, in a building or an automobile, is to provide comfort during either hot or cold weather.

A simple stylized diagram of the refrigeration cycle: 1) Condensing coil, 2) expansion valve, 3) evaporator coil, 4) compressor. Smt. V.Saritha Ellenki College of Engineering and Technology, JNTU, Hyderabad, Telangana, India.

In the refrigeration cycle, a heat pump transfers heat from a lower-temperature heat source into a highertemperature heat sink. Heat would naturally flow in the opposite direction. This is the most common type of air conditioning. A refrigerator works in much the same way, as it pumps the heat out of the interior and into the room in which it stands.

AIR CONDITIONING SYSTEM BASICS AND THEORIES

REFRIGERATION CYCLE



This cycle takes advantage of the way phase changes work, where latent heat is released at a constant temperature during a liquid/gas phase change, and where varying the pressure of a pure substance also varies its condensation/boiling point.

The most common refrigeration cycle uses an electric motor to drive a compressor. In an automobile, the compressor is driven by a belt over a pulley, the belt being driven by the engine's crankshaft (similar to the driving of the pulleys for the alternator, power steering, etc.). Whether in a car or building, both use electric fan motors for air circulation. Since evaporation occurs when heat is absorbed, and condensation occurs when heat is released, air conditioners use a compressor to cause pressure



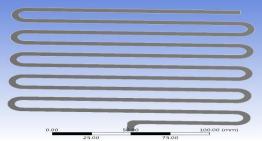
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changes between two compartments, and actively condense and pump a refrigerant around. A refrigerant is pumped into the evaporator coil, located in the compartment to be cooled, where the low pressure causes the refrigerant to evaporate into a vapor, taking heat with it. At the opposite side of the cycle is the condenser, which is located outside of the cooled compartment, where the refrigerant vapor is compressed and forced through another heat exchange coil, condensing the refrigerant into a liquid, thus rejecting the heat previously absorbed from the cooled space.

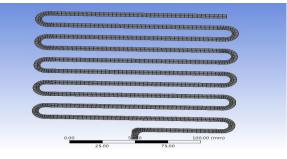
By placing the condenser (where the heat is rejected) inside a compartment, and the evaporator (which absorbs heat) in the ambient environment (such as outside), or merely running a normal air conditioners refrigerant in the opposite direction, the overall effect is the opposite, and the compartment is heated. This is usually called a heat pump, and is capable of heating a home to comfortable temperatures (25 °C; 70 °F), even when the outside air is below the freezing point of water (0 °C; 32 °F).

Cylinder un loaders are a method of load control used mainly in commercial air conditioning systems. On a semi-hermetic (or open) compressor, the heads can be fitted with un loaders which remove a portion of the load from the compressor so that it can run better when full cooling is not needed. Un loaders can be electrical or mechanical.

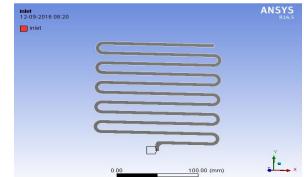
CFD ANALYSIS OF BI PROPELLENT EVAPORATOR R22 IMPORTED MODEL



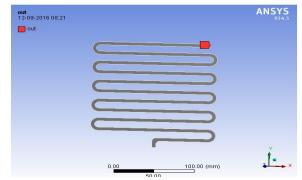
MESHED MODEL



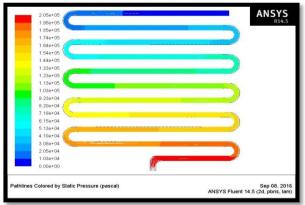
INLET



OUTLET



PRESSURE (PA)

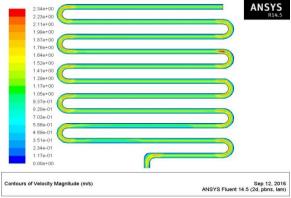


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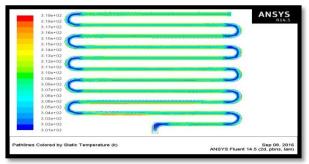


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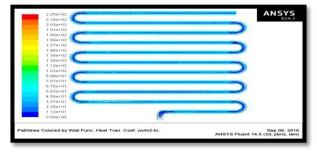
VELOCITY (M/SEC)



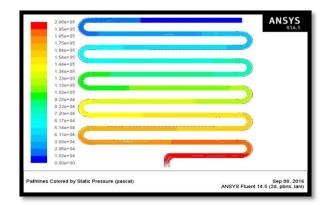
TEMPERATURE (K)



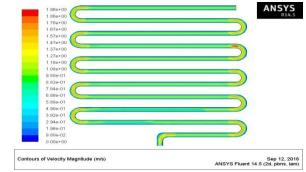
WALL FUNCTION HEAT TRANSFER COEFFICIENT (W/M2-K)



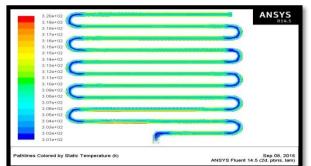
FLUID - R134A PRESSURE (PA)



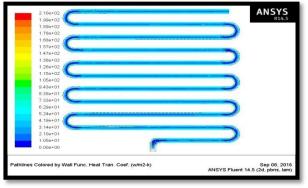
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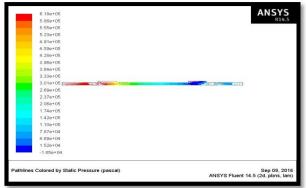
TEMPERATURE (K)



WALL FUNCTION HEAT TRANSFER COEFFICIENT (W/M2-K)



LOW PRESSURE COMPRESSOR PRESSURE (PA)

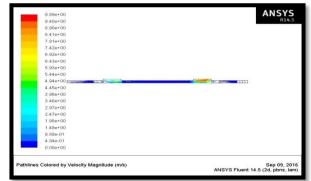


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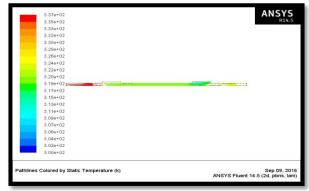


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VELOCITY (M/SEC)



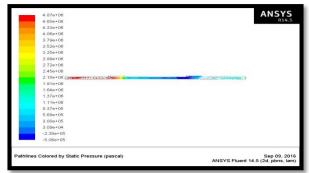
TEMPERATURE (K)



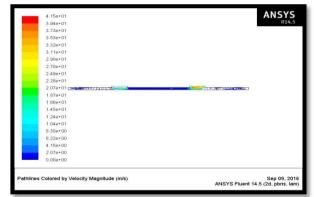
WALL FUNCTION HEAT TRANSFER COEFFICIENT (W/M2-K)

thlines Colored by Wall Func. Heat Tran. Coef. (w/m2-k)	Sep 09, 201 ANSYS Fluent 14.5 (2d, pbns, lam
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1.40e+01	
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4.20e+01	
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7.00e+01	
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9.80e+01	
1.12e+02	
1.26e+02	
1.40e+02t	
1.54e+02	
1.68e+02	
1.82e+02	
1.96e+02	
2.10e+02	
2.240+02	
2.380+02	
2.520+02	
2.66e+02	R14.5
2.80e+02	ANSYS

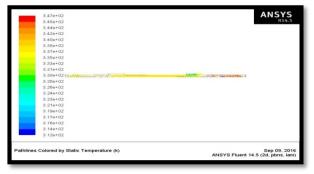
HIGH PRESSURE COMPRESSOR PRESSURE (PA)



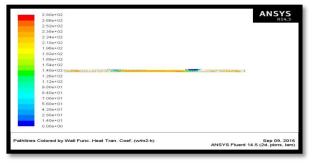
VELOCITY (M/SEC)



TEMPERATURE (K)

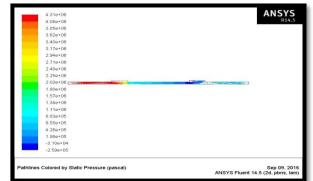


WALL FUNCTION HEAT TRANSFER COEFFICIENT (W/M2-K)



R134A

INLET BOUNDARY CONDITIONS PRESSURE (PA)

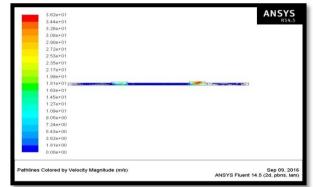


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VELOCITY (M/SEC)



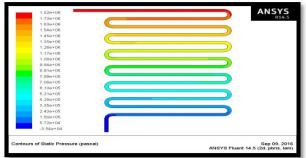
TEMPERATURE (K)

3.40e+02	ANSYS
3.36e+02	R14.5
3.33e+02	
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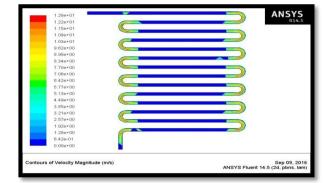
WALL FUNCTION HEAT TRANSFER COEFFICIENT (W/M2-K)

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LP CONDENSER PRESSURE (PA)



VELOCITY (M/SEC)



TEMPERATURE (K)

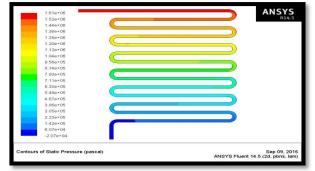
3.37e+02	ANSYS
3.33e+02	R14.5
3.280+02	
3.23e+02	
3.19e+02	
3.140+02	
3.10e+02	
3.05e+02	
3.00e+02	
2.96e+02	
2.91e+02	
2.87e+02	
2.82e+02	
2.77e+02	
2.73e+02	
2.68e+02	
2.63e+02	
2.59e+02	
2.540+02	
2.50e+02	
2.45e+02	
Contours of Static Temperature (k)	Sep 09, 201
	ANSYS Fluent 14.5 (2d, pbns, lan

WALL FUNCTION HEAT TRANSFER COEFFICIENT (W/M2-K)

ANSYS	1	-	2.17e+02
R14.5			2.06e+02
			1.95e+02
		- Martine -	1.84e+02
		F	1.74++02
		<u> </u>	1.63e+02
			1.52e+02
		-	1.41e+02
			1.30e+02
			1.19e+02
		C	1.08e+02
			9.76e+01
			8.68e+01
		C	7.59e+01
		and the second s	6.51e+01
		F	5.42e+01
		C	4.34e+01
			3.25e+01
		F .	2.17e+01
			1.08e+01
		N	0.00e+00
Sep 09, 2016 S Fluent 14.5 (2d, pbns, lam)	ANSYS Flue	Tran. Coef. (w/m2-k)	ontours of Wall Func. Heat

R134A

PRESSURE (PA)

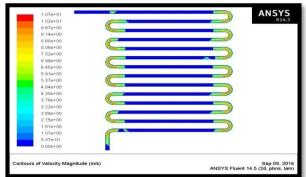


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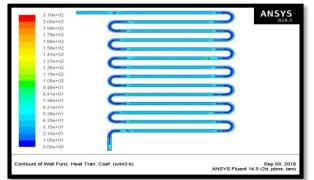
VELOCITY (M/SEC)



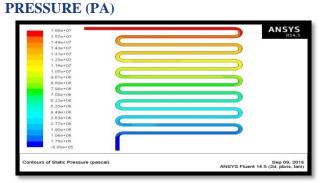
TEMPERATURE (K)

ANS	+02
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Sep 0 ANSYS Fluent 14.5 (2d, pb	c Temperature (k)

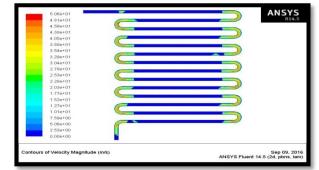
WALL FUNCTION HEAT TRANSFER COEFFICIENT (W/M2-K)



H.P. CONDENSER R22



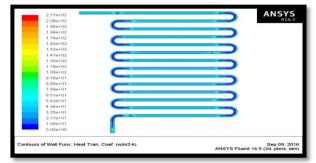
VELOCITY (M/SEC)



TEMPERATURE (K)

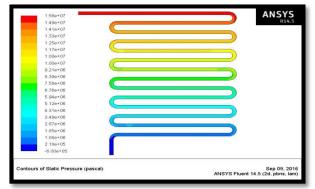
	le+02		ANSYS
3.4	le+02		 R14.5
3.4	le+02		
3.4	le+02		
3.3	e+02		
3.3	le+02		
3.3	e+02		
3.2	le+02		
3.2	ie+02		
3.2	10+02		
3.1	le+02		
3.1	ie+02		
3.14	le+02		
3.1	10+02		
3.0	le+02		
3.0	ie+02	And the second	
3.0	le+02		
2.9	le+02		
2.9	ie+02		
2.9	le+02		
2.9	le+02		
Contours of Sta	tic Temperature (k)		 Sep 09, 201 14.5 (2d, pbns, lan

WALL FUNCTION HEAT TRANSFER COEFFICIENT (W/M2-K)



R134A

INLET BOUNDARY CONDITIONS PRESSURE (PA)

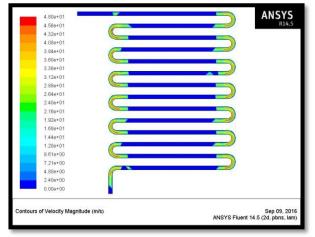


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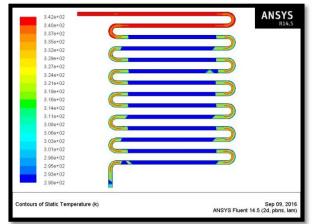


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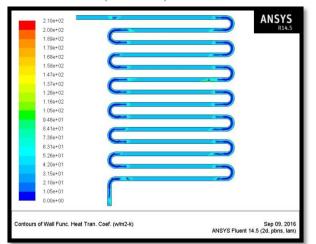
VELOCITY (M/SEC)



TEMPERATURE (K)



WALL FUNCTION HEAT TRANSFER COEFFICIENT (W/M2-K)



RESULTS AND DISCUSSIONS EVAPORATOR

Flu ids	Pres sure (Pa)	Tempe rature (K)	Vel ocit y (M/ Sec)	Wall Funct ion Heat Trans fer Coeff icient (W/M 2-K)	Mass Flow Rate (Kg/Sec)	Tota 1 Heat Tran sfer Rate (W)
R2 2	2051 09.5	319.00 63	2.34 3	224.8 694	- 0.00245 8	10.1 69
R1 34 A	2055 86.7	320.31 97	1.95 90	209.5 591	- 0.00010 001358	- 5.32 6416

COMPRESSOR LOW PRESSURE COMPRESSOR

Flui ds	Press ure (Pa)	Tempe rature (K)	Velo city (M/S ec)	Wall Functi on Heat Transf er Coeffi cient (W/M 2-K)	Mas s Flow Rate (Kg/ Sec)	Total Heat Trans fer Rate (W)
R22	6181 20	337.2	9.88 96	279.9 3	- 0.27 691	1042 5.809
R13 4A	5624 48.2	335.97	8.35 9242	271.2 692	0.04 99	7898. 7874

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HIGH PRESSURE COMPRESSOR

Flui ds	Pres sure (Pa)	Tempe rature (K)	Velo city (M/S ec)	Wall Functi on Heat Trans fer Coeffi cient (W/M 2-K)	Mass Flow Rate (Kg/S ec)	Total Heat Trans fer Rate (W)
R2 2	4866 064	346.99	41.4 7	279.9 091	- 0.436 2	1019 77.76
R1 34 A	4310 480	339.53 23	36.2 1076	271.2 3	- 1.172 5349	7796 5.76

CONDENSER LOW PRESSURE CONDENSER

Flui ds	Pres sure (Pa)	Tempe rature (K)	Velo city (M⁄ Sec)	Wall Functi on Heat Transf er Coeffi cient (W/M 2-K)	Mass Flow Rate (Kg/S ec)	Total Heat Trans fer Rate (W)
R22	1817	337.32	12.8	216.9	0.003	167.4
	868	9	313	602	6125	5703
R13	1606	335.12	10.7	210.2	0.001	72.31
4A	323		4	4	159	25

Flui đs	Press ure (Pa)	Tempe rature (K)	Velo city (M/ Sec)	Wall Functi on Heat Trans fer Coeffi cient (W/M 2-K)	Mass Flow Rate (Kg/ Sec)	Total Heat Trans fer Rate (W)
R2 2	1.65e +7	348.85	50	216.9 6	- 0.00 6256	- 259.4 1074
R1 34 A	1.575 2e+7	342.48	48.0 41	210.2 4	- 0.00 80	466.7 89

HIGH PRESSURE CONDENSER

CONCLUSION

In recent years, the escalating cost of energy has drawn much more attention on improving the energy efficiency of super market operations. In a supermarket refrigeration system consume a large amount of energy in maintaining chilled and frozen food. Meanwhile a HVAC (heating, ventilating and air conditioning) system is used to assure thermal comfort for occupants and suitable climatic conditions for refrigerated cases.

In this thesis, the thermal performance of combined air conditioning and refrigeration unit will be analyzed by CFD. 3D model and assembly of the combined air conditioning and refrigeration unit done in creo.

By observing above CFD results comparing of two refrigerants R22 and R134a of combined refrigeration unit R134a is getting low temperature comparatively with R22 for various condensers are placed for various applications. While, AC supply point of view we placed condenser at high pressure compressor in this R134a shows less temperature $(17^{\circ}c)$ is comes out from the condenser, in the same way temperature(- $25^{\circ}C$) is observed at condenser which is placed at low pressure condenser which is supplied to refrigerators placed in super markets.



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