# Design and Analysis of Black Hawk UH-60 Rotor Blade Using Composite Materials 

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## ABSTRACT

In depth literature survey on the main rotor blade of helicopters was carried out, and Black Hawk UH-60 has chosen for design, development and analysis. The blade configuration details including the plan form, taper, root joint and profiles was studied. The various load cases were analyzed.

The main scope of this project is to Design a hollow composite rotor blade using Carbon Fibre Reinforced Plastic (CFRP) with different stacking sequences. The required thicknesses at various span wise locations to compensate the developed loads were calculated using strength of material approach. Finally the model is analyzed using ANSYS software to know the resulting displacements, stresses and mode shapes.

## INTRODUCTION

## Rotor blade

The rotor blades of a helicopter are airfoils that provide aerodynamic forces when exposed to a relative motion of air across their surface. The rotational motion of the rotor hub initiated by the helicopter engine develops this relative motion, as well as forward, sideward and backward flight. While developing aerodynamics lift and drag forces, structural loads occur on the blades along their span and across their chord. A helicopter's rotor is generally made of two or more rotor blades. . Each main rotor is mounted on a vertical mast over the top of the helicopter, as opposed to a helicopter tail rotor, which connects through a combination of drive shaft(s) and gearboxes along the tail boom.

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## LITERATURE SURVEY

Specifications of Black Hawk UH-60
Crew: 2 pilots (flight crew) with 2 crew chiefs/gunners Length: 64 ft 10 in ( 19.76 m )
Fuselage width: 7 ft 9 in ( 2.36 m )
Rotor diameter: 53 ft 8 in ( 16.36 m )
Height: 16 ft 10 in ( 5.13 m )
Disc area: $2,260 \mathrm{ft}^{2}\left(210 \mathrm{~m}^{2}\right)$
Empty weight: $10,624 \mathrm{lb}(4,819 \mathrm{~kg})$
Loaded weight: $22,000 \mathrm{lb}(9,980 \mathrm{~kg})$
Max. Takeoff weight: $23,500 \mathrm{lb}(10,660 \mathrm{~kg})$
Power plant: $2 \times$ General Electric T700-GE-701C turbo shaft, $1,890 \mathrm{hp}(1,410 \mathrm{~kW})$ each

## Performance:

Never exceed speed: 193 knots ( $222 \mathrm{mph} ; 357 \mathrm{~km} / \mathrm{h}$ )
Maximum speed: 159 KN ( $183 \mathrm{mph} ; 295 \mathrm{~km} / \mathrm{h}$ )
Cruise speed: 150KN ( $173 \mathrm{mph} ; 278 \mathrm{~km} / \mathrm{h}$ )
Combat radius: 368 mi ( 320 nautical miles; 590 km )


Fig 2.1 Orientation of Black Hawk UH-60
Like airplane wings, early rotor blades had a main spar, ribs and a covering of fabric. The blades were symmetrical because this shape offered a good lift-todrag ratio and a stable center of pressure, and they were easy to build. This was to be the design and shape of rotor blades for many years. The early blades

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were a blend of several types of wood, each adding properties the other materials lacked.

## Material properties

Selection of material: Carbon Fiber
Material Description

| Material: | T700s 12k $3900-2$ |
| :---: | :---: |
| Form: | Plain weave fabric prepreg, 3 tows per inch, fiber areal weight of $193 \mathrm{~g} \mathrm{gm}^{2}$, typical cured |
|  | resin content of $35 \%$.-66\%, typical cured ply thickness of $0.0073-0.0079$ inches. |
| Procesing: | Autoclave cure, $350^{\circ} \mathrm{F}, 85$ psi, $3{ }^{\circ} \mathrm{F} /$ minuteramp rate, 2 hours |
| General Sup | Information |

T700 fibers are continuous, standard modulus, no twist carbon filaments made from a PAN precursor, sufface treated to improve handling characteristics and structural properties. Filament count is 12,000 iliaments tow. Typical tensile modulus is $34 \times 10^{6}$ Fiber: pssi.

Typical tensile strength is 700,000 psi
3900-2 is an toughened epoxyresign

## DESIGN OF ROTOR BLADE

Blade Configuration
Overall configuration and key data which is required to replicate a blade are collected. Literature survey has been carried out on the configuration and load cases.


Fig 3.1 Blade Configuration

| Configuration | Actual |
| :---: | :---: |
| Length | 5 m |
| Root Chord | 298.28 mm |
| Tip Chord | 238.04 mm |
| Surface Area Blade | 1.4 sq.mts |
| Surface Area Disc | 210 sq.mts |
| Sweep Angle L.E | 20 deg |

Table 3.1 Configuration of Blade

Various points on the Blade are located by station numbers


Fig 3.2 Blade span-wise stations

## Chord line and maximum thickness.

The airfoil chosen was a SC1094R8 and SC1095, which is one of the more favorable of an extended series of related airfoils developed. The SC1095 airfoil and the SC1094 R8 airfoil, modified from the SC1095 by adding droop at the leading edge. The effect of the nose droop was to extend the SC1095 chord from 20.76 in. to 20.965 in., thereby reducing the airfoil thickness from 9.5 percent to 9.4 percent (hence the change in the section nomenclature). The addition of nose droop also rotated the mean chord line by -1 deg, the two airfoil sections are compared on a nondimensional basis

There is a linear transition between the SC1094 R8 and SC1095airfoil sections. The SC1095 section is then used to the blade tip.

The root airfoil measured 295.28 mm chord length while the wing tip airfoil measured 238.04 mm chord length.

## SC1095



## SC1094 R8

Fig 3.3 Chord length and Maximum thickness of airfoils at each stations are noted.

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| Station | Chord | Maximum |
| :---: | :---: | :---: |
| 0 | 295.28 | 27.25 |
| 1 | 296.998 | 27.408 |
| 2 | 285.59 | 26.356 |
| 3 | 274.017 | 25.288 |
| 4 | 271.047 | 25.013 |
| 5 | 270.846 | 24.995 |
| 6 | 270.645 | 24.976 |
| 7 | 270.445 | 24.719 |
| 8 | 270.244 | 24.701 |
| 9 | 270.043 | 24.683 |
| 10 | 269.842 | 24.664 |
| 11 | 267.114 | 24.415 |
| 12 | 272.216 | 24.881 |
| 13 | 247.832 | 22.871 |
| 14 | 238.042 | 21.968 |
| 15 | 228.549 | 21.092 |

Table 3.2 Chord and Thickness (at C/4) at each station

Chord-wise and Span-wise thickness calculation

| S.no | Station | Chord <br> Length | $\mathbf{5 5 \%}$ of <br> Chord | Flexural <br> Strength | Bending <br> Moment (N- <br> mm) | $\mathbf{M / F}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0 | 295.28 | 162.404 | 466.6666667 | 1059600 | 2270.571429 |
| 2 | 1 | 296.998 | 163.3489 | 466.6666667 | 908630 | 1947.064286 |
| 3 | 2 | 285.59 | 157.0745 | 466.6666667 | 685330 | 1468.564286 |
| 4 | 3 | 274.017 | 150.70935 | 466.6666667 | 609420 | 1305.9 |
| 5 | 4 | 271.047 | 149.07585 | 466.6666667 | 583031 | 1249.352143 |
| 6 | 5 | 270.846 | 148.9653 | 466.6666667 | 572433 | 1226.642143 |
| 7 | 6 | 270.645 | 148.85475 | 466.6666667 | 551030 | 1180.778571 |
| 8 | 7 | 270.445 | 148.74475 | 466.6666667 | 540830 | 1158.921429 |
| 9 | 8 | 270.244 | 148.6342 | 466.6666667 | 532731 | 1141.566429 |
| 10 | 9 | 270.043 | 148.52365 | 466.6666667 | 432741 | 927.3021429 |
| 11 | 10 | 269.842 | 148.4131 | 466.6666667 | 413372 | 885.7971429 |
| 12 | 11 | 267.114 | 146.9127 | 466.6666667 | 399945 | 857.025 |
| 13 | 12 | 272.216 | 149.7188 | 466.6666667 | 384066 | 822.9985714 |
| 14 | 13 | 247.832 | 136.3076 | 466.6666667 | 362725 | 777.2678571 |
| 15 | 14 | 238.042 | 130.9231 | 466.6666667 | 350730 | 751.5642857 |
| 16 | 15 | 228.549 | 125.70195 | 466.6666667 | 348712 | 747.24 |

Table 3.3 Chord-wise and Span-wise thickness calculation

| Station | $\mathbf{t}$ |
| :---: | :---: |
| 0 | 1.0675 |
| 1 | 0.899 |
| 2 | 0.7292 |
| 3 | 0.7043 |
| 4 | 0.689 |
| 5 | 0.677 |
| 6 | 0.652 |
| 7 | 0.647 |
| 8 | 0.638 |
| 9 | 0.517 |
| 10 | 0.4935 |
| 11 | 0.488 |
| 12 | 0.4495 |
| 13 | 0.51 |
| 14 | 0.5352 |
| 15 | 0.58 |

## Blade Design

The drawing of SC-1094R8 and SC-1095 airfoil is produced using AutoCAD by plotting the coordinates. The airfoils at each station are designed in such a way that, the rib is at the required pitch angle


Fig 3.8 Blade designed using Composite Thickness

## ANALYSIS OF ROTOR BLADE

## Finite Element Analysis

For the sake of validating results ANSYS software has chosen. The material properties of each resin system is obtained based on their data sheets and tested results of neat resin casting (provided by CPDC) and using rule of mixtures the required properties $\left(\mathrm{E}_{1}, \mathrm{E}_{2}, \mathrm{v}_{12}, \mathrm{v}_{23}\right.$, $\mathrm{G}_{12}, \mathrm{G}_{23}$ ) are calculated. These values are given as inputs to the software and various tests are carried out according to the actually tested specimens.

## Material Properties of Carbon fiber

Carbon T700S properties
$\mathrm{E}_{11}=139.27 \mathrm{GPA}$
$\mathrm{E}_{22}=7.843 \mathrm{GPA}$
$\mathrm{E} 33=7.843$
$v_{12}=0.255$
$v_{23}=0.38$
$v_{31}=0.255$
$\mathrm{G}_{12}=3.802 \mathrm{GPA}$
$\mathrm{G}_{23}=2.951 \mathrm{GPa}$
$\mathrm{G}_{31}=3.802 \mathrm{GPa}$
Density $=1565 \mathrm{~kg} / \mathrm{m}^{3}$

## Results

The imported model from AUTOCAD into ANSYS was meshed using mesh tool command resulting in fine mesh which is shown in the following figures.


Fig 4.1: Imported model from AUTOCAD


Fig4.2: Total meshed body


Fig 4.3: Zoomed out mesh

## Case I

In the first case the considered stacking sequence is [90/0/90/0/90/0/90/0/90/0/90] which is a symmetrical stacking sequence.

The obtained results from static analysis are shown in the following figures


Fig 4.4: Selected stacking sequence for the first case


Fig 4.5: Displacement plot for the first case


Fig 4.6: Von Misses stress plot for the first case

The modal analysis has resulted in the following mode shapes


Fig 4.7: Mode-1 for the first case


Fig 4.8: Mode-2 for the first case


Fig 4.9: Mode - 3 for the first case


Fig 4.10: Mode - 4 for the first case


Fig 4.11: Mode - 5 for the first case


Fig 4.12: Mode - 6 for the first case

## Case II

In the second case the considered stacking sequence is [45/-45/45/-45/45/0/45/-45/45/-45/45] which is a symmetrical stacking sequence

The obtained results from static analysis are shown in the following figures


Fig 4.13: Stacking sequence for Second case

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Fig 4.14: Displacement plot for the second case


Fig 4.15: Von Misses stress plot for the second case

The modal analysis has resulted in the following mode shapes


Fig 4.16: Mode - 1 for the second case


Fig 4.17: Mode - 2 for the second case


Fig 4.18: Mode - 3 for the second case


Fig 4.19: Mode - 4 for the second case


Fig 4.20: Mode - 5 for the second case


Fig 4.21: Mode - 6 for the second case

Case III
In the third case the considered stacking sequence is [-45/45/90/0/90/0/90/0/90/45/-45] which is a symmetrical stacking sequence.

The obtained results from static analysis are shown in the following figures


Fig 4.22: Stacking sequence for third case


Fig 4.23: Displacement plot for the third case


Fig 4.24: Von Misses stress plot for the third case

The modal analysis has resulted in the following mode shapes


Fig 4.25: Mode-1 for the third case


Fig 4.26: Mode - 2 for the third case


Fig 4.27: Mode - 3 for the third case


Fig 4.28: Mode - 4 for the third case


Fig 4.29: Mode - 5 for the third case


Fig 4.30: Mode - 6 for the third case

## CONCLUSION AND FUTURE SCOPE

Conclusion
After observing the results from ANSYS the following conclusions are derived

1. The maximum displacement for case-1 stacking sequence is $=13.89 \mathrm{~mm}$
2. The maximum stress for case-1 stacking sequence is $=54.27 \mathrm{MPA}$
3. The maximum displacement for case-2 stacking sequence is $=54.94 \mathrm{~mm}$
4. The maximum stress for case-2 stacking sequence is $=106.149 \mathrm{MPA}$
5. The maximum displacement for case-3 stacking sequence is $=17 \mathrm{~mm}$
6. The maximum stress for case-3 stacking sequence is $=42 \mathrm{MPA}$

Comparing the modal analysis for all the cases, the obtained natural frequencies are

|  | case- $\mathbf{1}$ | case $-\mathbf{2}$ | case $-\mathbf{3}$ |
| :---: | :---: | :---: | :---: |
| Mode -1 | $1.70 \mathrm{E}-05$ | $8.49 \mathrm{E}-06$ | $1.53 \mathrm{E}-05$ |
| Mode -2 | $9.63 \mathrm{E}-05$ | $5.14 \mathrm{E}-05$ | $9.09 \mathrm{E}-05$ |
| Mode -3 | $1.32 \mathrm{E}-04$ | $6.64 \mathrm{E}-05$ | $1.21 \mathrm{E}-04$ |
| Mode -4 | $2.00 \mathrm{E}-04$ | $1.41 \mathrm{E}-04$ | $2.43 \mathrm{E}-04$ |
| Mode -5 | $2.44 \mathrm{E}-04$ | $2.72 \mathrm{E}-04$ | $4.13 \mathrm{E}-04$ |
| Mode -6 | $4.25 \mathrm{E}-04$ | $3.98 \mathrm{E}-04$ | $4.55 \mathrm{E}-04$ |



Comparing the modal analysis for all the cases, the obtained deflections for different modes are

|  | case-1 | case-2 | case-3 |
| :---: | :---: | :---: | :---: |
| Mode - 1 | $3.11 \mathrm{E}-04$ | $3.09 \mathrm{E}-04$ | $3.08 \mathrm{E}-04$ |
| Mode - 2 | $3.03 \mathrm{E}-04$ | $3.18 \mathrm{E}-04$ | $3.14 \mathrm{E}-04$ |
| Mode - 3 | $0.306-3$ | $3.09 \mathrm{E}-04$ | $3.08 \mathrm{E}-04$ |
| Mode - 4 | $5.35 \mathrm{E}-04$ | $3.31 \mathrm{E}-04$ | $3.44 \mathrm{E}-04$ |
| Mode - 5 | $3.18 \mathrm{E}-04$ | $3.40 \mathrm{E}-04$ | $4.80 \mathrm{E}-04$ |
| Mode -6 | $3.60 \mathrm{E}-04$ | $3.21 \mathrm{E}-04$ | $3.28 \mathrm{E}-04$ |



After observing the results it is found that quasi isotropic stacking sequence is better for the manufacture of rotor blades which has resulted in less stresses i.e. compared to all other selected stacking sequences. It also resulted in reasonable minimum displacement i.e. 17 mm compared to other stacking sequences.

## Future scope.

A current goal of Defense is to use adaptive or "smart" materials that adjust the airfoil shape to reduce rotor vibration..


Possible future shape

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In the current design, the number of layers of composites is fixed, and each layer has the same layer thickness. That is, the layer thickness is equal to the total thickness divides by the number of layers. However, because the feasible values of layer thickness are certain discrete numbers, this assumption possibly results in the infeasible layer thickness. A better way is to choose layer thickness and the number of layers as design variables. If the fiber orientations of all layers are 96 dependent design variables, the number of variables of the optimization problem is changeable. The applicable algorithm for such problem is very difficult to find in literature.

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